

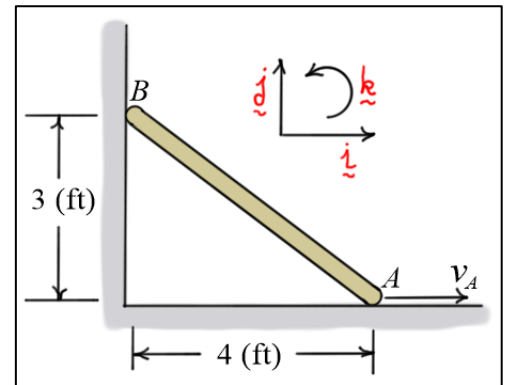
Elementary Dynamics

Exercises #7 – Two-Dimensional Rigid Body Kinematics

1. Bar AB rests against a vertical wall as shown. At the instant shown, the velocity of A is **constant** $\underline{v}_A = 9\hat{i}$ (ft/s). At this instant, find: a) \underline{v}_B the velocity of B and $\underline{\omega}_{AB}$ the angular velocity of AB , and b) \underline{a}_B the acceleration of B and $\underline{\alpha}_{AB}$ the angular acceleration of AB .

Answers:

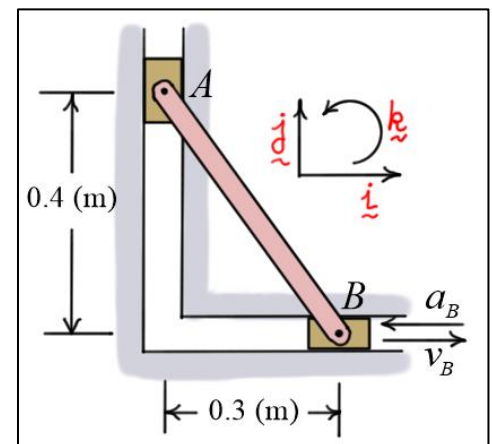
- a) $\underline{v}_B \approx -12\hat{j}$ (ft/s) and $\underline{\omega}_{AB} \approx 3\hat{k}$ (rad/s)
 b) $\underline{a}_B \approx -75\hat{j}$ (ft/s²) and $\underline{\alpha}_{AB} \approx 12\hat{k}$ (rad/s²)



2. Bar AB has its ends constrained to move in the horizontal and vertical slots. At the instant shown, the point B has velocity $\underline{v}_B = 2\hat{i}$ (m/sec) and acceleration $\underline{a}_B = -5\hat{i}$ (m/sec²). At this instant, find: a) $\underline{\omega}_{AB}$ the angular velocity of AB and \underline{v}_A the velocity of A , and b) $\underline{\alpha}_{AB}$ the angular acceleration of AB and \underline{a}_A the acceleration of A .

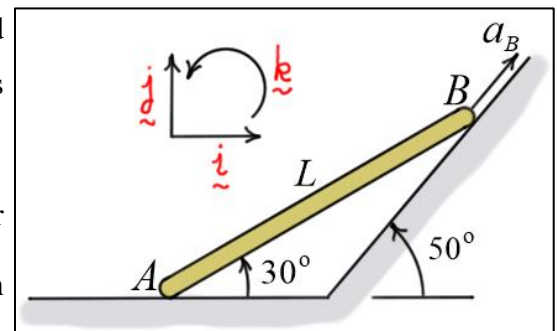
Answers:

- a) $\underline{\omega}_{AB} \approx 5\hat{k}$ (rad/s) and $\underline{v}_A \approx -1.5\hat{j}$ (m/s);
 b) $\underline{\alpha}_{AB} \approx 6.25\hat{k}$ (rad/s²) and $\underline{a}_A \approx -11.9\hat{j}$ (m/s²)

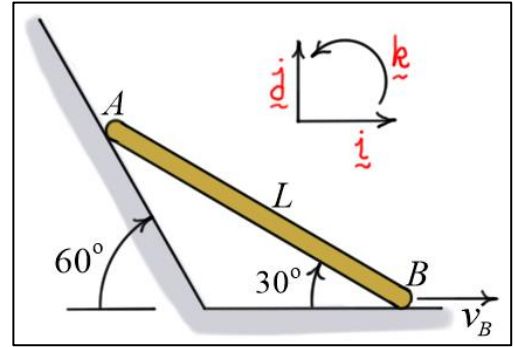


3. The ends of bar AB slide along the horizontal and inclined surfaces. At the instant shown, the angular velocity of AB is $\underline{\omega}_{AB} = 2\hat{k}$ (rad/s), and the acceleration of end B is $a_B = 1$ (ft/s²) up the inclined plane. At this instant, find $\underline{\alpha}_{AB}$ the angular acceleration of AB and \underline{a}_A the acceleration of end A . The length of AB is $L = 5$ (ft).

Answers: $\underline{\alpha}_{AB} \approx 2.49\hat{k}$ (rad/s²) and $\underline{a}_A \approx 24.2\hat{i}$ (ft/s²)



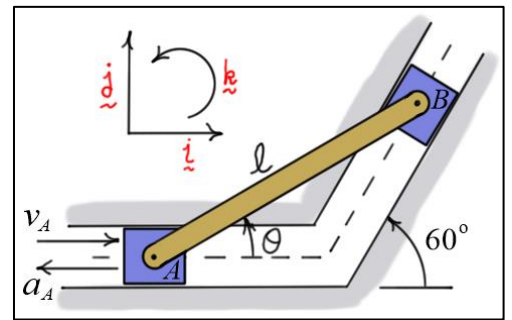
4. The ends of bar AB of length $L = 10$ (in) slide along the horizontal and inclined surfaces as shown. At the instant shown, end B has velocity $\underline{v}_B = 20\hat{i}$ (in/sec). a) Using the **relative velocity equation**, find \underline{v}_A the velocity of A and $\underline{\omega}_{AB}$ the angular velocity of AB at this instant. b) Repeat part (a) using the concept of **instantaneous centers** of zero velocity.



Answers:

$$|\underline{v}_A| \approx 20 \text{ (in/s)}; \underline{v}_A \approx 10\hat{i} - 17.3\hat{j} \text{ (in/s)}; \underline{\omega}_{AB} \approx 2\hat{k} \text{ (rad/s)}$$

5. The figure shows bar AB of length $\ell = 2$ (ft) whose ends slide along the horizontal and inclined surfaces. At the instant shown, angle $\theta = 30$ (deg), the velocity of A is $\underline{v}_A = 5\hat{i}$ (ft/s), and the acceleration of A is $\underline{a}_A = -10\hat{i}$ (ft/s²). At this instant, find: a) $\underline{\omega}_{AB}$ the angular velocity of AB and \underline{v}_B the velocity of B , and b) $\underline{\alpha}_{AB}$ the angular acceleration of AB and \underline{a}_B the acceleration of B .

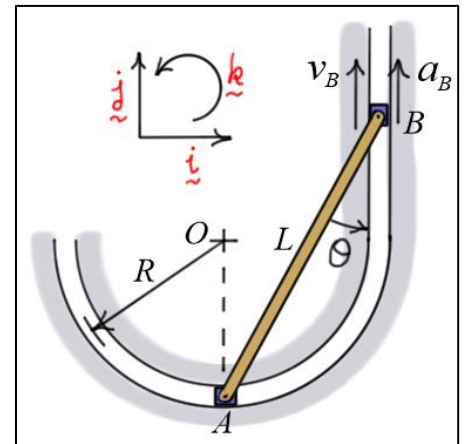


Answers:

$$\text{a) } \underline{\omega}_{AB} \approx 2.5\hat{k} \text{ (rad/s) and } \underline{v}_B \approx 5(\cos(60)\hat{i} + \sin(60)\hat{j}) \text{ (ft/s)}$$

$$\text{b) } \underline{\alpha}_{AB} \approx -8.61\hat{k} \text{ (rad/s}^2\text{) and } \underline{a}_B \approx -24.4(\cos(60)\hat{i} + \sin(60)\hat{j}) \text{ (ft/s}^2\text{)}$$

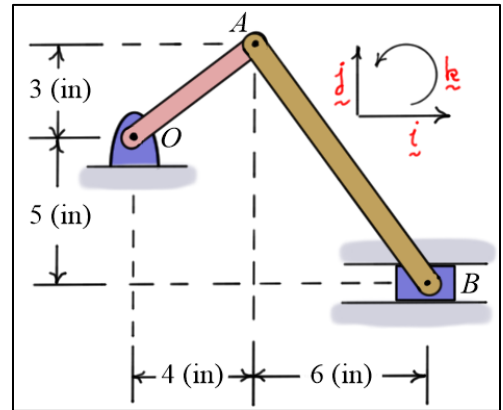
6. At the instant shown, end A of bar AB moves along a circular slot while end B moves along the straight slot. The radius of the circular slot is $R = 0.2$ (m), the length of the bar is $L = 0.4$ (m), and the angle $\theta = 30$ (deg). At the instant shown, the velocity of B is $\underline{v}_B = 5\hat{j}$ (m/s) and the acceleration of B is $\underline{a}_B = 10\hat{j}$ (m/s²). a) Using the concept of **instantaneous centers** of zero velocity, find $\underline{\omega}_{AB}$ the angular velocity of AB and \underline{v}_A the velocity of A . b) Using the **relative acceleration equation**, find $\underline{\alpha}_{AB}$ the angular acceleration of AB and \underline{a}_A the acceleration of A .



Answers:

$$\text{a) } \underline{\omega}_{AB} \approx 25\hat{k} \text{ (rad/s) and } \underline{v}_A \approx 8.66\hat{i} \text{ (m/s)}; \text{ b) } \underline{\alpha}_{AB} \approx -742\hat{k} \text{ (rad/s}^2\text{) and } \underline{a}_A \approx -132\hat{i} + 375\hat{j} \text{ (m/s}^2\text{)}$$

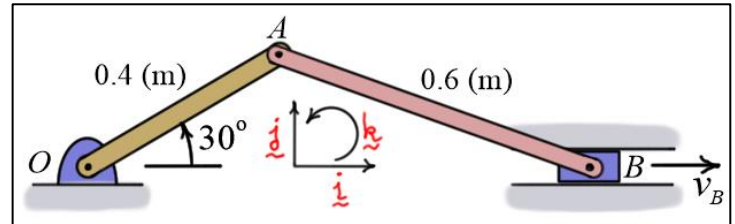
7. The figure shows a slider-crank mechanism OAB . Crank OA is driven at a **constant** angular velocity of $\omega_{OA} = -10\hat{k}$ (rad/sec). a) Using the **relative velocity equation**, find ω_{AB} the angular velocity of the connecting bar AB and v_B the velocity of the slider B . b) Repeat part (a) using the concept of **instantaneous centers** of zero velocity.



Answers:

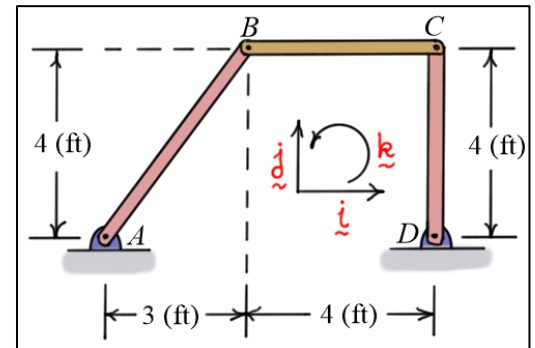
$$\omega_{AB} \approx 6.67\hat{k} \text{ (rad/s)} \text{ and } v_B \approx 83.3\hat{i} \text{ (in/s)}$$

8. The figure shows slider-crank mechanism OAB . At the instant shown, the velocity of B is $v_B = 3\hat{i}$ (m/s). Using the concept of **instantaneous centers** of zero velocity, find ω_{AB} the angular velocity of the connecting rod AB and v_A the velocity of A at this instant.



Answers: $\omega_{AB} \approx 5.7\hat{k}$ (rad/s) and $v_A \approx 3.72(\sin(30)\hat{i} - \cos(30)\hat{j})$

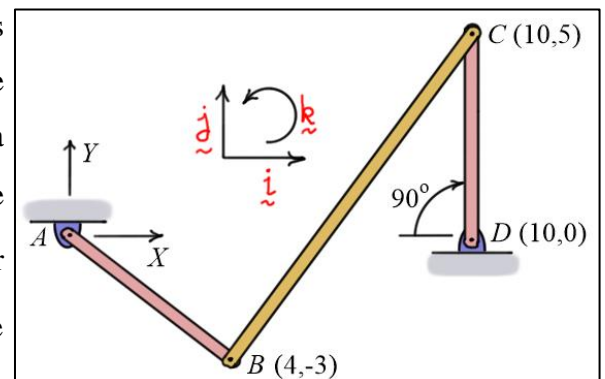
9. At the instant shown, the **angular velocity** of bar AB of the four-bar mechanism $ABCD$ is $\omega_{AB} = 10\hat{k}$ (rad/s). Using the concept of **instantaneous centers** of zero velocity, find ω_{BC} and ω_{CD} the angular velocities of bars BC and CD .



Answers:

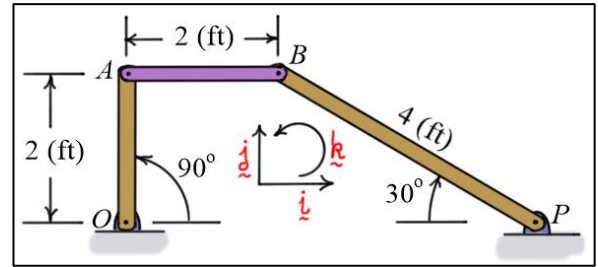
$$\omega_{BC} \approx -7.5\hat{k} \text{ (rad/s)}; \omega_{CD} \approx 10\hat{k} \text{ (rad/s)}$$

10. The figure shows a four-bar mechanism $ABCD$. Point A is located at the origin, and points B , C and D have the coordinates shown. At the instant shown, link AB has a **constant** angular velocity of $\omega_{AB} = 9\hat{k}$ (rad/s). a) Using the **relative velocity equation**, find ω_{BC} and ω_{CD} the angular velocities of links BC and CD . b) Repeat part (a) using the concept of **instantaneous centers** of zero velocity.



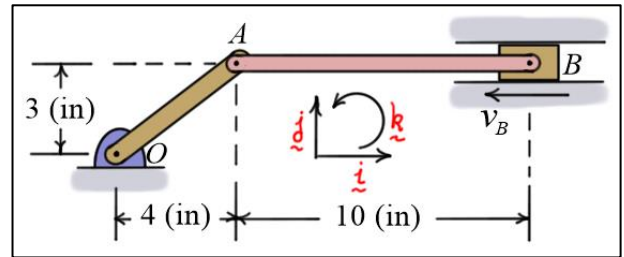
Answers: $\omega_{BC} \approx -6\hat{k}$ (rad/s) and $\omega_{CD} \approx -15\hat{k}$ (rad/s)

11. The figure shows a four-bar mechanism $OABP$. At the instant shown, the angular velocity of the crank OA is $\omega_{OA} = 5\mathbf{k}$ (rad/s). Using the concept of *instantaneous centers* of zero velocity, find ω_{AB} the angular velocity of connecting bar AB .



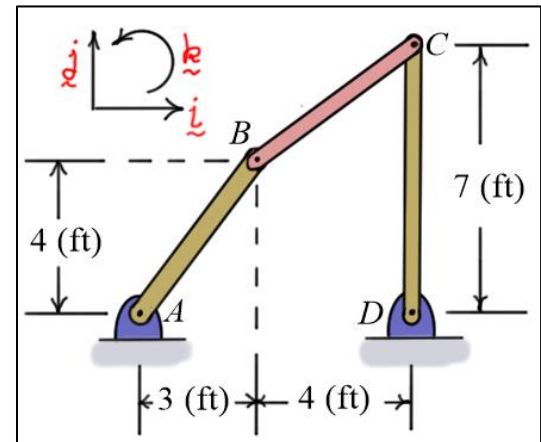
Answer: $\omega_{AB} \approx -8.66\mathbf{k}$ (rad/s)

12. The figure shows a slider-crank mechanism OAB . At the instant shown, the velocity of slider B is $v_B = -6\mathbf{i}$ (in/s).
 a) Using the *relative velocity equation*, find ω_{AB} the angular velocity of connecting rod AB and ω_{OA} the angular velocity of crank OA . b) Repeat part (a) using the concept of *instantaneous centers* of zero velocity.



Answers: $\omega_{AB} \approx -0.8\mathbf{k}$ (rad/s) and $\omega_{OA} \approx 2\mathbf{k}$ (rad/s)

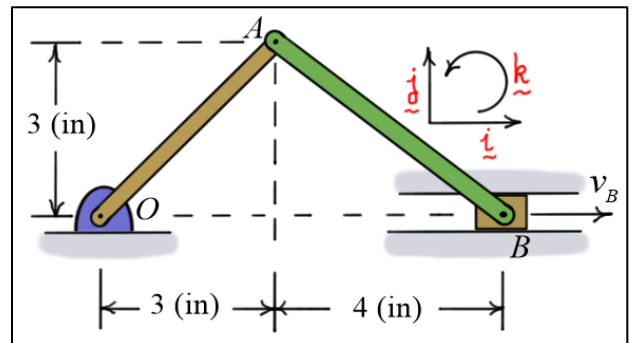
13. At the instant shown, the angular velocity of bar AB of the four-bar mechanism $ABCD$ is $\omega_{AB} = 10\mathbf{k}$ (rad/s). Using the *relative velocity equation*, find a) v_B the velocity of point B , and b) ω_{BC} and ω_{CD} the angular velocities of bars BC and CD .



Answers:

- a) $v_B \approx -40\mathbf{i} + 30\mathbf{j}$ (ft/s)
 b) $\omega_{BC} \approx -7.5\mathbf{k}$ (rad/s); $\omega_{CD} \approx 2.5\mathbf{k}$ (rad/s)

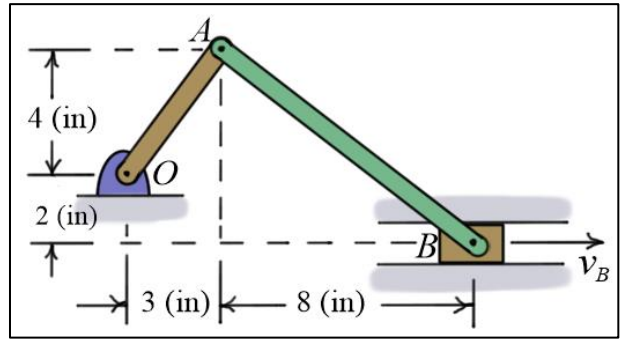
14. The figure shows a slider-crank mechanism OAB . The slider B moves at a *constant velocity* $v_B = 10.5\mathbf{i}$ (in/s). At the instant shown, find: a) ω_{OA} and ω_{AB} the angular velocities of the two bars, and b) α_{OA} and α_{AB} the angular accelerations of the bars.



Answers:

- a) $\omega_{OA} = -2\mathbf{k}$ (rad/s), $\omega_{AB} = +1.5\mathbf{k}$ (rad/s); b) $\alpha_{OA} \approx -3.25\mathbf{k}$ (rad/s²), $\alpha_{AB} \approx 3.75\mathbf{k}$ (rad/s²)

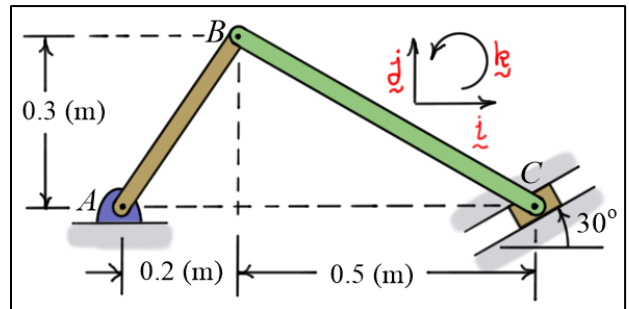
15. The figure shows a slider-crank mechanism OAB . The slider has **constant** velocity $v_B = 20\hat{i}$ (in/sec). At the instant shown, find: a) ω_{OA} and ω_{AB} the angular velocities of OA and AB , and b) α_{OA} and α_{AB} the angular accelerations of OA and AB .



Answers:

- a) $\omega_{OA} = -3.2\hat{k}$ (rad/sec) and $\omega_{AB} = 1.2\hat{k}$ (rad/sec);
 b) $\alpha_{OA} \approx -2.88\hat{k}$ (rad/s²) and $\alpha_{AB} \approx 5.12\hat{k}$ (rad/s²)

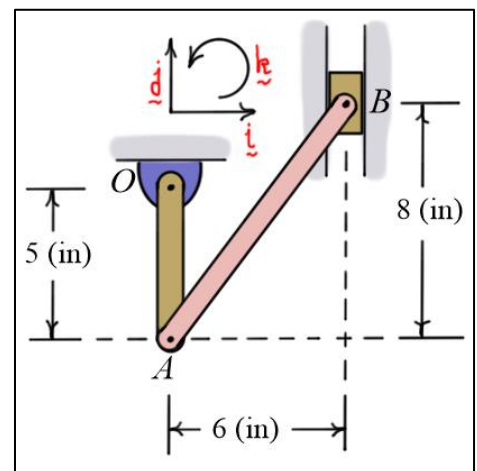
16. The figure shows a slider-crank mechanism ABC . Bar AB has a **constant** angular velocity of $\omega_{AB} = -10\hat{k}$ (rad/sec). Using the equations for relative velocity and acceleration, find: a) ω_{BC} the angular velocity of BC and v_C the velocity of C , and b) α_{BC} the angular acceleration of BC and a_C the acceleration of C .



Answers:

- a) $\omega_{BC} \approx 11.4\hat{k}$ (rad/s) and $v_C \approx 7.42(\cos(30)\hat{i} + \sin(30)\hat{j})$ (m/s)
 b) $\alpha_{BC} \approx -178\hat{k}$ (rad/s²) and $a_C \approx -160(\cos(30)\hat{i} + \sin(30)\hat{j})$ (m/s²)

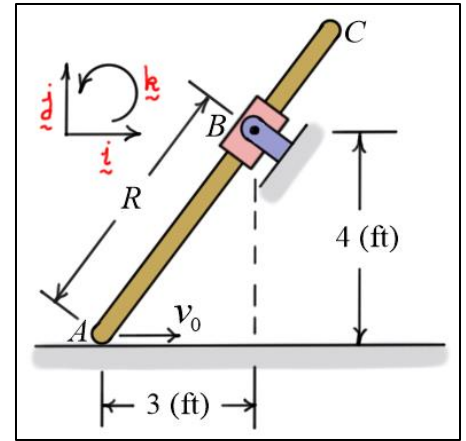
17. The figure shows a slider-crank mechanism OAB . Crank OA is driven at a **constant** rate of $\omega_{OA} = 8\hat{k}$ (rad/sec). At the instant shown, find: a) ω_{AB} the angular velocity of AB and v_B the velocity of B , and b) α_{AB} the angular acceleration of AB and a_B the acceleration of B .



Answers:

- a) $\omega_{AB} \approx 5\hat{k}$ (rad/sec) and $v_B \approx 30\hat{j}$ (in/s)
 b) $\alpha_{AB} \approx -18.8\hat{k}$ (rad/s²) and $a_B \approx 7.5\hat{j}$ (in/s²)

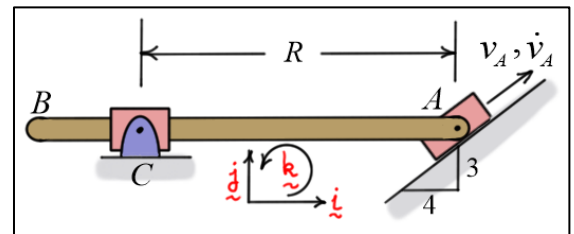
18. Bar AC rests on the horizontal plane at A and slides through a *smooth collar* at B . The collar rotates freely to allow the bar to rotate as A moves at a *constant* speed of $v_0 = 10 \hat{i}$ (ft/s). The variable distance from A to B is R . At the instant shown, find: a) ω_{AC} the angular velocity of AC and \dot{R} the time rate of change of the distance R , and b) α_{AC} the angular acceleration of AC and \ddot{R} the time rate of change of \dot{R} .



Answers:

- a) $\omega_{AC} \approx 1.6 \hat{k}$ (rad/s); $\dot{R} \approx -6.0$ (ft/s)
 b) $\alpha_{AC} \approx 3.84 \hat{k}$ (rad/s²); $\ddot{R} \approx 12.8$ (ft/s²)

19. Bar AB slides through the collar at C while its end A moves up the inclined plane. The collar rotates freely to allow the bar to rotate as A moves up the plane. At the instant shown, the velocity and acceleration of A are $v_A = 10$ (ft/s) and $\dot{v}_A = 5$ (ft/s²), and the variable distance from C to A is $R = 2$ (ft).

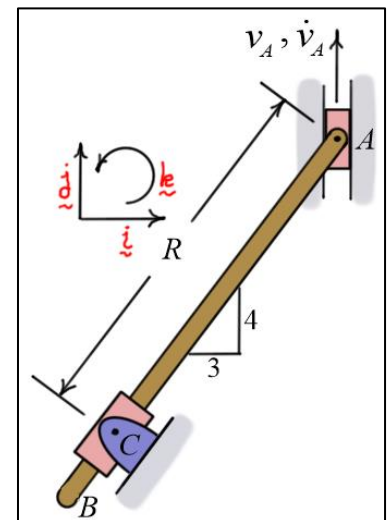


At this instant, find: a) ω_{AB} the angular velocity of AB and \dot{R} the time rate of change of R , and b) α_{AB} the angular acceleration of AB and \ddot{R} the time rate of change of \dot{R} .

Answers:

- a) $\omega_{AB} \approx 3 \hat{k}$ (rad/s) and $\dot{R} \approx 8$ (ft/s); b) $\alpha_{AB} \approx -22.5 \hat{k}$ (rad/s²) and $\ddot{R} \approx 22$ (ft/s²)

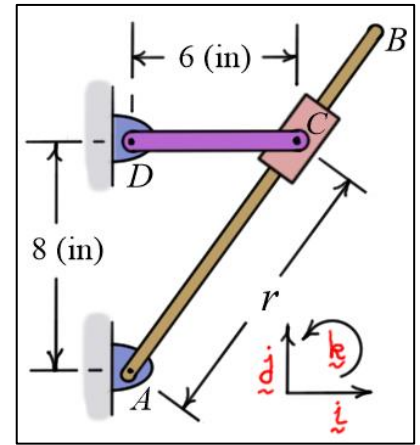
20. Bar AB slides through a collar at C while its end A moves in the vertical slot. The collar rotates freely to allow the bar to rotate as A moves up. At the instant shown, the velocity and acceleration of A are $v_A = 12.5 \hat{j}$ (ft/s) and $\dot{v}_A = 5 \hat{j}$ (ft/s²), and the variable distance from C to A is $R = 5$ (ft). At this instant, find a) ω_{AB} the angular velocity of AB and \dot{R} the time rate of change of R , and b) α_{AB} the angular acceleration of AB and \ddot{R} the time rate of change of \dot{R} .



Answers: a) $\omega_{AB} = 1.5 \hat{k}$ (rad/s), $\dot{R} = 10$ (ft/s);

- b) $\alpha_{AB} \approx -5.4 \hat{k}$ (rad/s²) and $\ddot{R} \approx 15.3$ (ft/s²)

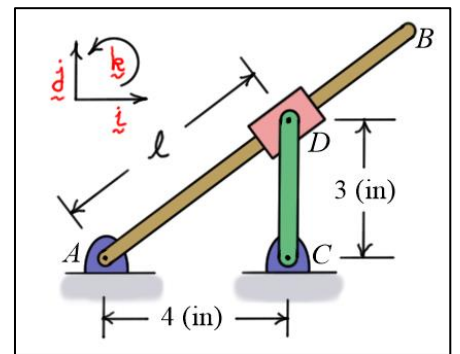
21. The system shown consists of two bars AB and CD connected by a collar at C . Bar AB is free to slide through the collar as it rotates at a **constant** rate of $\omega_{AB} = 18 \hat{k}$ (rad/s), and bar CD is pinned to the collar at C . The variable distance from A to C is represented by the symbol r . At the instant shown, find: a) ω_{CD} the angular velocity of CD and \dot{r} the time rate of change of the distance r , and b) α_{CD} the angular acceleration of CD and \ddot{r} the time rate of change of \dot{r} .



Answers:

a) $\omega_{CD} \approx 50 \hat{k}$ (rad/s) and $\dot{r} \approx 20$ (ft/s); b) $\alpha_{CD} \approx -933 \hat{k}$ (rad/s²) and $\ddot{r} \approx -853$ (ft/s²)

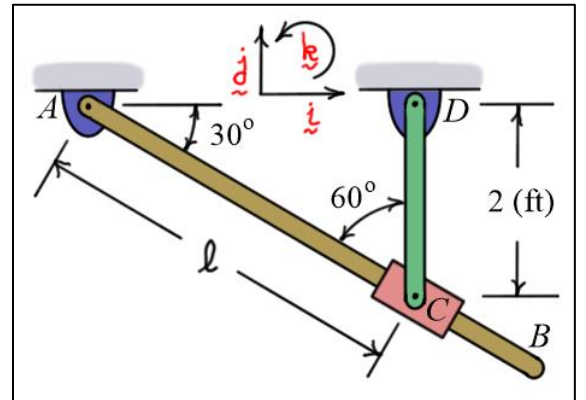
22. The system shown consists of two bars AB and CD and a collar at D . The collar is pinned to bar CD and is free to slide along and rotate with bar AB . The variable distance between A and D is ℓ . Bar CD rotates at a **constant rate** $\omega_{CD} = 10 \hat{k}$ (rad/s). At the instant shown, find: a) ω_{AB} the angular velocity of AB and $\dot{\ell}$ the time rate of change of ℓ , and b) α_{AB} the **angular acceleration** of AB , and $\ddot{\ell}$ the time derivative of $\dot{\ell}$.



Answers:

a) $\omega_{AB} = 3.6 \hat{k}$ (r/s) and $\dot{\ell} = -24$ (in/s); b) $\alpha_{AB} \approx -13.4 \hat{k}$ (rad/s²) and $\ddot{\ell} \approx -115$ (in/s²)

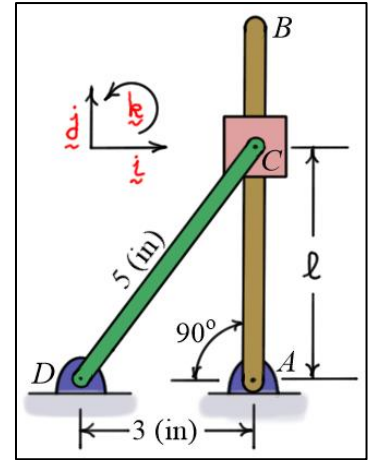
23. The system shown consists of two bars AB and CD and a collar at C . The collar is pinned to bar CD and is free to slide along and rotate with bar AB . The **variable length** between A and C is ℓ . Bar CD rotates at a **constant rate** of $\omega_{CD} = 5 \hat{k}$ (rad/s). At the instant shown when $\ell = 4$ (ft), find: a) ω_{AB} the angular velocity of AB and $\dot{\ell}$ the time derivative of ℓ , and b) α_{AB} the angular acceleration of AB and $\ddot{\ell}$ the time derivative of $\dot{\ell}$.



Answers:

a) $\omega_{AB} \approx 1.25 \hat{k}$ (rad/s) and $\dot{\ell} \approx 8.66$ (ft/s); b) $\alpha_{AB} \approx 5.41 \hat{k}$ (rad/s²) and $\ddot{\ell} \approx -18.8$ (ft/s²)

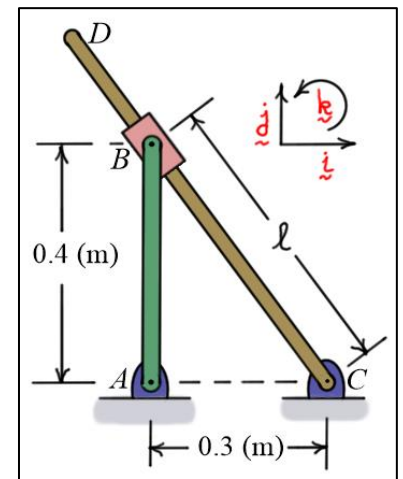
24. The system shown consists of two bars AB and CD connected by a collar at C . Bar AB is free to slide through the collar as it rotates, and bar CD is pinned to the collar. Length ℓ represents the **variable distance** from A to C . At the instant shown, length $\ell = 4$ (in) and the angular velocity of AB is **constant** $\omega_{AB} = 5\mathbf{k}$ (rad/s). At this instant, find: a) ω_{CD} the angular velocity of CD and $\dot{\ell}$ the first derivative of the distance ℓ , and b) α_{CD} the angular acceleration of CD and $\ddot{\ell}$ the time derivative of $\dot{\ell}$.



Answers:

a) $\omega_{CD} \approx 5\mathbf{k}$ (rad/s) and $\dot{\ell} \approx 15$ (in/s); b) $\alpha_{CD} \approx 18.8\mathbf{k}$ (rad/s²) and $\ddot{\ell} \approx 56.3$ (in/s²)

25. The system shown consists of two bars AB and CD connected by a collar at B . Bar CD is free to slide through the collar as it rotates, and bar AB is pinned to the collar. The length ℓ represents the **variable distance** from C to B . At the instant shown, length $\ell = 0.5$ (m) and the angular velocity of AB is **constant** $\omega_{AB} = 10\mathbf{k}$ (rad/s). At this instant, find: a) ω_{CD} the angular velocity of CD and $\dot{\ell}$ the time derivative of the length ℓ , and b) α_{CD} the angular acceleration of CD and $\ddot{\ell}$ the time derivative of $\dot{\ell}$.



Answers:

a) $\omega_{CD} \approx 6.4\mathbf{k}$ (rad/s) and $\dot{\ell} \approx 2.4$ (m/s); b) $\alpha_{CD} \approx -13.4\mathbf{k}$ (rad/s²) and $\ddot{\ell} \approx -11.5$ (m/s²)