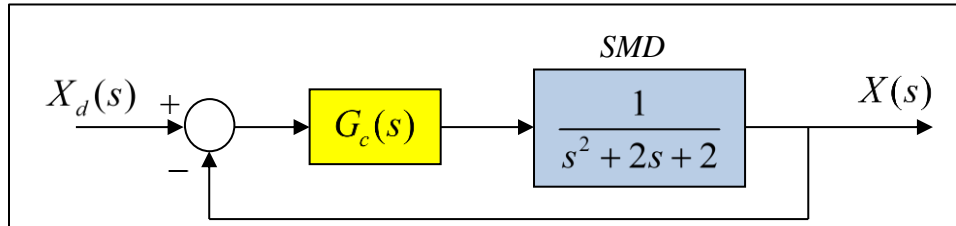


## Introductory Motion and Control

### Root Locus Design of a Phase-Lag Compensator for a Spring-Mass-Damper (SMD) Positioning System

To illustrate the *root locus design* of a *phase-lag compensator*, consider the following SMD positioning system controlled by the compensator  $G_c(s)$ . Here,  $X_d(s)$  and  $X(s)$  are the *desired* and *actual* positions of the mass.



Using proportional control ( $G_c(s) = K$ ), *large* gains are required to control *steady-state error* for a step input. Unfortunately, *large gains* produce *undesirable, oscillatory* closed-loop response. Below, a phase-lag compensator is designed to lower the steady-state error without introducing highly oscillatory behavior.

**Problem:** Design a phase-lag compensator so the closed-loop system has a *steady-state step position error*  $e_{ss} = 1 - x_{ss} < 0.1$  and *damping factors* for the complex poles of  $\zeta \geq 0.4$ . Plot the step response of the resulting closed-loop system.

#### Root Locus Design

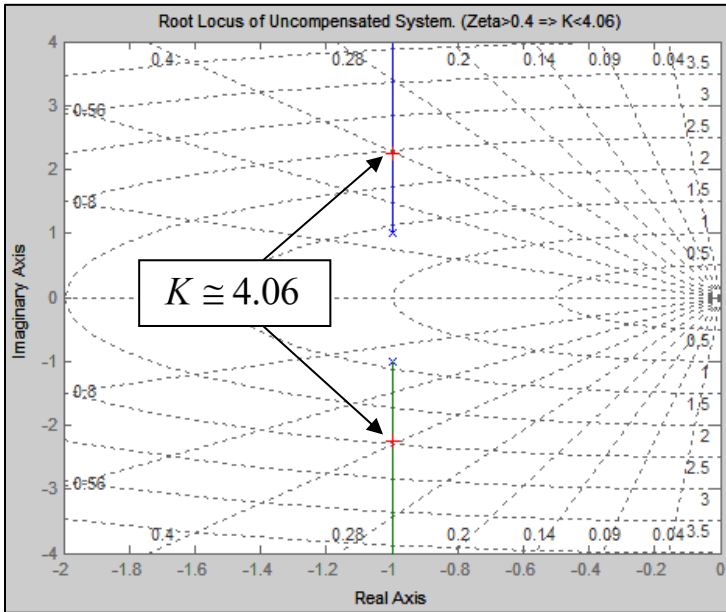
**Step 1:** Examine *RL diagram* of *uncompensated system*. Find the *gain* required to satisfy the *steady-state error* requirement.

The root locus diagram of the uncompensated system is very simple. The poles of  $GH(s)$  are  $-1 \pm 1j$ . For  $K > 0$  the roots move to infinity along the asymptotes at  $\sigma_A = -1$ . For  $K < 0$  the roots move into the break point at  $-1$  and then move along the positive and negative real axis.

For  $\zeta \geq 0.4$ , we use MATLAB to show that  $K \leq 4.06$ . See diagram below. Using  $K = 4.06$ , the steady-state error for the uncompensated system is

$$e_{ss} = \lim_{s \rightarrow 0} \left[ \frac{1}{1 + GH(s)} \right] = \frac{1}{1 + (4.06/2)} = 0.33$$

This is clearly higher than the specified value of 0.1. To lower the steady-state error to 0.1, we require  $K = 18$ .



Root Locus of Uncompensated System. For the damping ratio to be greater than 0.4, the system gain must be less than 4.06.

If  $K = 18$  is used, a *phase-lag compensator* can be added to increase the system damping. This will also *increase the settling time* of the system.

**Step 2: Evaluate** how the *compensator* changes the *RL diagram*.

Here, the loop transfer function  $GH(s) = \frac{K \alpha (s + z)}{(s + p)(s^2 + 2s + 2)}$ , so the system has asymptotes

at  $\phi_A = \pm 90$  (deg) that intersect the real axis at  $\sigma_A = \frac{2(-1) - p + z}{2}$ . In a phase-lag compensator,

$z > p$ , so the asymptotes will be *moved to the right*. However, they will not be moved far, because the *pole* and *zero* are generally located *close to each other*.

**Step 3: Try different pole-zero combinations** to see effect on RL diagram

Based on the ratio of the uncompensated system gain ( $K \approx 4$ ) and the desired gain ( $K \approx 18$ ), start by picking a pole close to the origin (say  $p = 0.1$ ) and setting

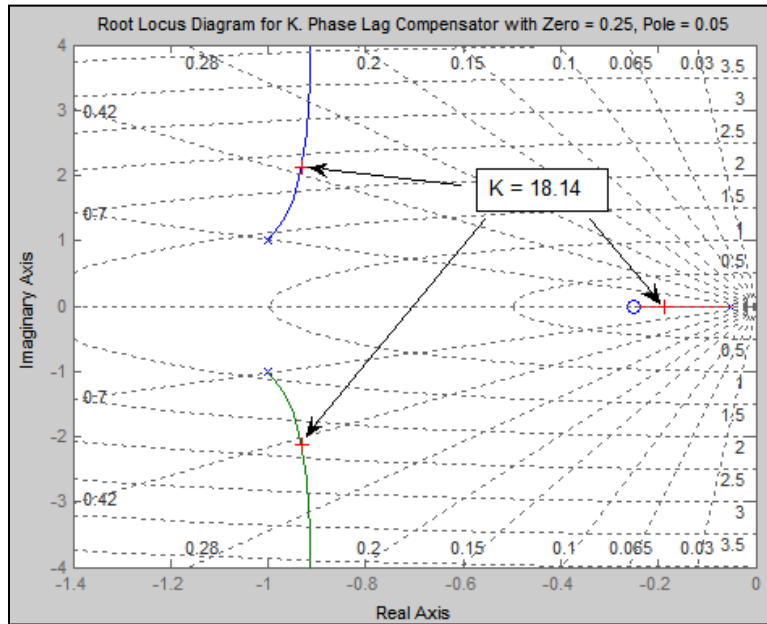
$$\alpha = K_{\text{uncompensated}} / K_{\text{desired}} = 4 / 18 \approx 0.2 \quad \text{and} \quad z = p / \alpha = 0.1 / 0.2 = 0.5$$

**Two questions must now be answered:** 1) Can roots be found with  $\zeta > 0.4$ ? 2) Can a *large enough value for K* be chosen so the steady-state error is small? From the root locus diagram, if  $\zeta \geq 0.4$ , then the maximum  $K$  value is approximately 16. **After some iteration** (moving the pole/zero combination closer to the imaginary axis), the following results were found.

For  $p=0.05$  and  $z=0.25$ , the loop transfer function of the compensated system is

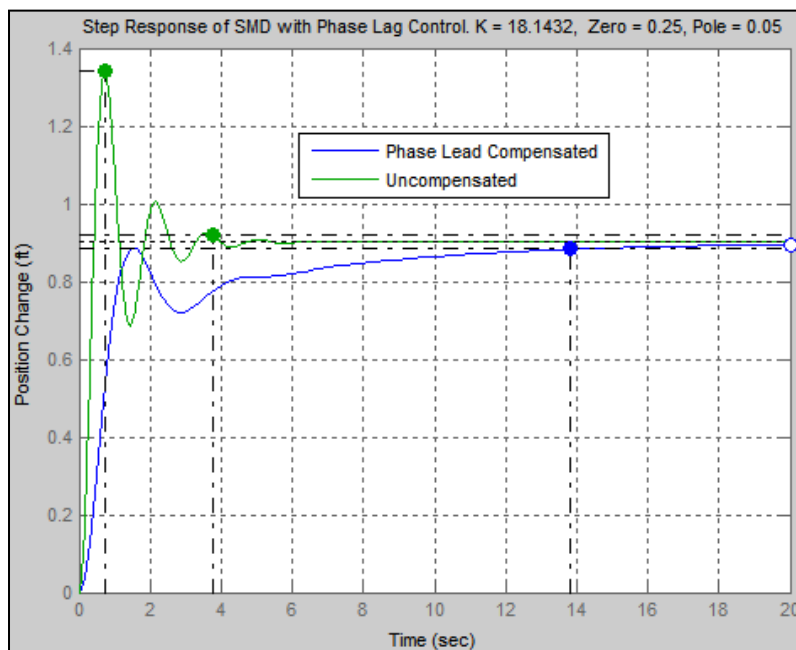
$$GH(s) = \frac{0.2K(s+0.25)}{(s+0.05)(s^2+2s+2)}$$

and the root locus diagram yields  $K \approx 18$  when the complex poles have  $\zeta \approx 0.4$ . See the diagram below.



**Step 4: Check the step response.**

The step response of the *uncompensated system* with gain  $K \approx 18.14$  (same steady-state error as the compensated system) shows a **large overshoot** (49%), low damping, and a **settling time** of approximately 3.8 seconds, while the step response of the compensated system shows no overshoot, higher damping, and a settling time around 13.8 seconds.



If some *overshoot is acceptable* (allowing smaller  $\zeta$ ), the *gain can be increased* in the above design to yield the results shown below for a gain of  $K \approx 27.44$ . With this gain the system has about 10% overshoot and a settling time of 11 seconds. Note also that with the larger gain, the steady state error is smaller ( $e_{ss} \approx 0.07$ ).

