

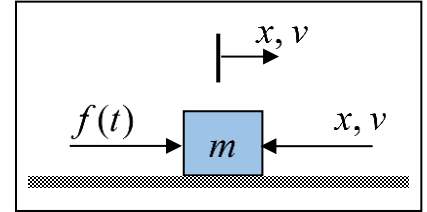
Introductory Control Systems

Proportional Control of a First Order System

Example System: Simple Speed Control System

Consider a *car* of *mass* m traveling along a road with *wind resistance* (proportional to the speed of the car) as shown in the diagram. Applying Newton's 2nd law in the direction of travel and neglecting friction, write

$$\sum F = f(t) - cv = m\dot{v} \quad \Rightarrow \quad m\dot{v} + cv = f(t)$$

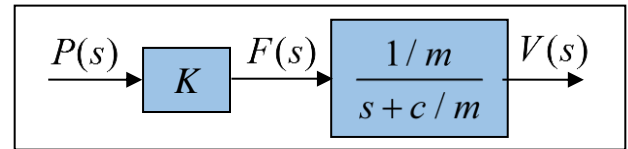


Using *Laplace transforms*, the *transfer function* for the system is

$$\frac{V}{F}(s) = \frac{1/m}{s + c/m} \tag{1}$$

Open-Loop, Proportional Speed Control

Now consider *proportional, open-loop speed control* of the car as indicated in the block diagram.



The system *transfer function* is

$$\frac{V}{P}(s) = \frac{K/m}{s + c/m} \tag{2}$$

The *final value* due to a *unit step* input $P(s) = 1/s$ is $v_{ss} = (K/m)/(c/m) = K/c$.

Fig. 1 shows the step response of this system for $K = 300, 600,$ and 900 using the parameters shown in Eq. (3) below. Note the value of K affects the *magnitude* of the response, but it *does not affect* how long the car takes to reach a new final speed. That is, it *does not affect* the system's *settling time*.

$$\begin{matrix} m = 100 \text{ slugs} \\ c = 20 \text{ (lb-s/ft)} \end{matrix} \tag{3}$$

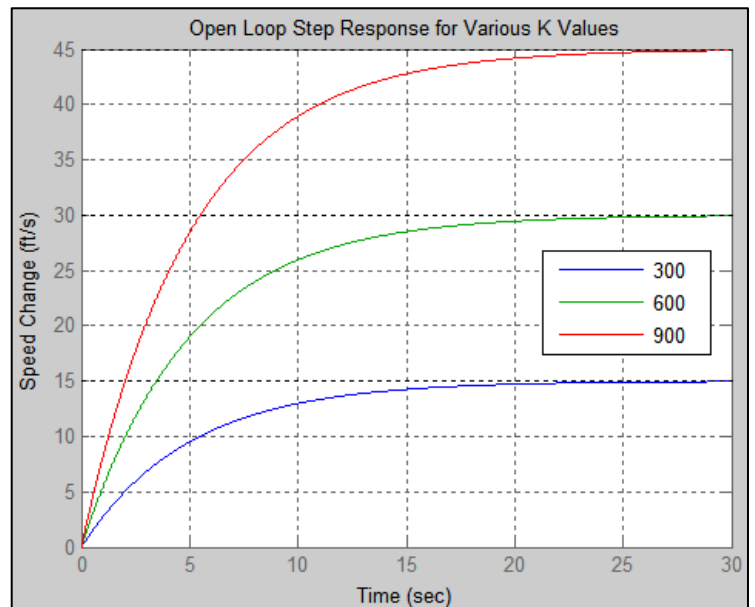
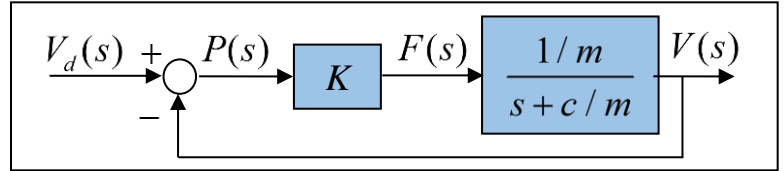


Fig. 1 Open-Loop Step Response for Various K Values

Closed-Loop, Proportional Speed Control

Finally, consider the *proportional, closed-loop speed control* of the car as indicated in the block diagram. The transfer function of this system is



$$\frac{V}{V_d}(s) = \frac{K/m}{s + c/m + K/m} = \frac{K/m}{s + (c + K)/m} \quad (4)$$

The system input is V_d the desired speed, and the output is the actual speed. The final value due to a *step input* $V_d(s) = 1/s$ is $v_{ss} = (K/m) / ((c + K)/m) = K / (c + K)$. Fig. 2 shows the step response of the system for $K = 300, 600,$ and 900 . Note that the value of K *affects both* the *magnitude* of the response and the *time* it takes the car to reach a new final speed. *Higher* values of K give *shorter* settling times.

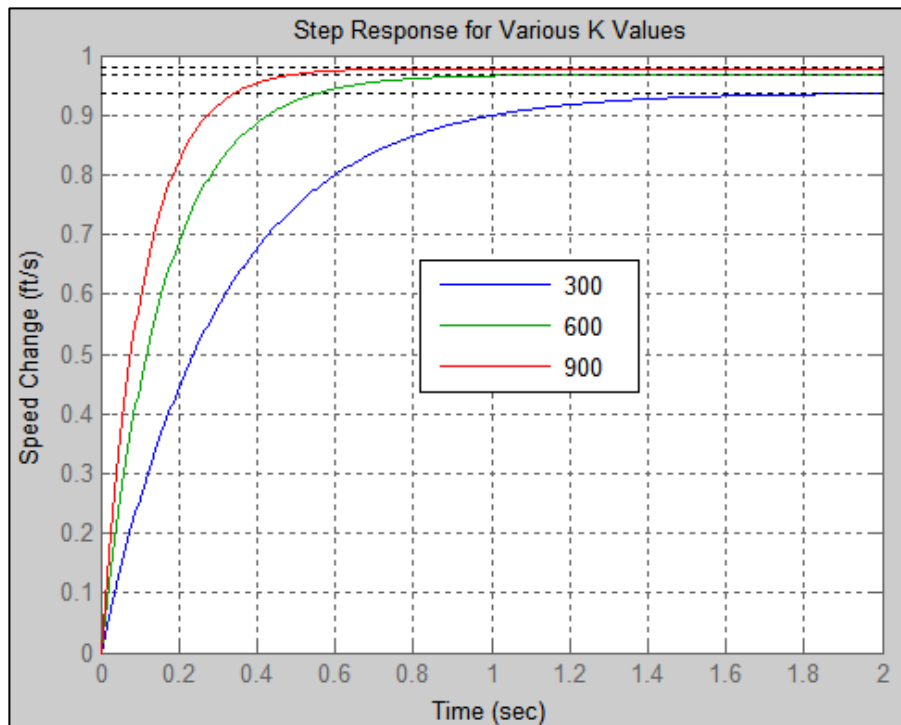


Fig. 2 Closed-Loop Step Response for Various K Values

In theory, the value of K could be *increased* further to make the steady state response (v_{ss}) closer to the commanded value ($=1$ (ft/s)) and the settling time smaller and smaller. However, as these changes are made, the *force* required to move the car becomes higher and higher. Fig. 3 shows the driving force $f(t)$ associated with the *unit step responses* shown in Fig. 2. Clearly, higher velocity commands and higher gains will cause the *forces* to eventually become *unrealistic*.

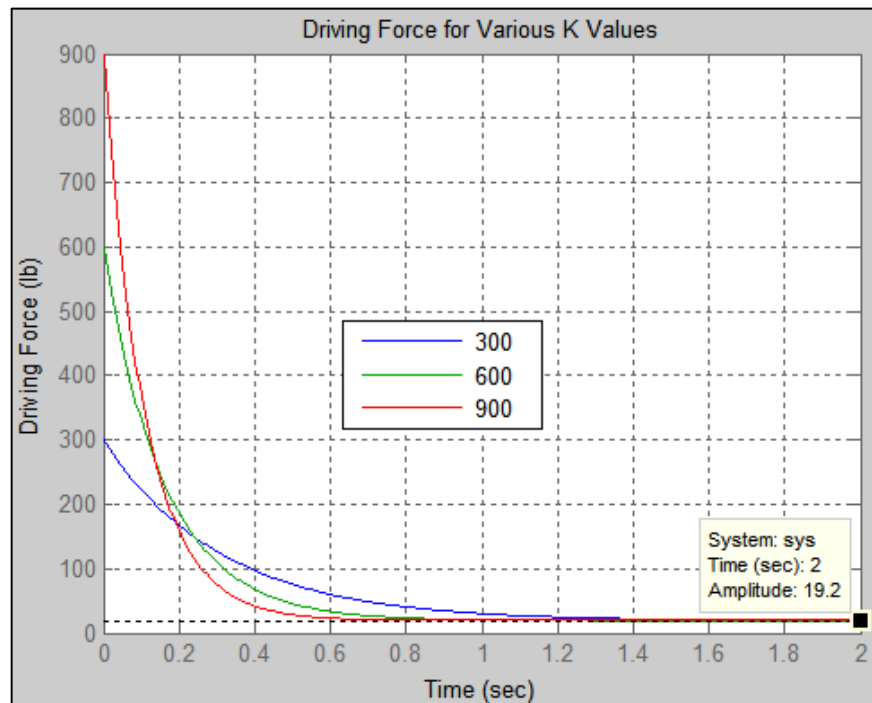


Fig. 3 Driving Force for Closed-Loop Step Response for Various Gains