

Intermediate Dynamics

Degrees of Freedom of Mechanical Systems

System Configuration and Generalized Coordinates

The *configuration* of a mechanical system is defined as the *position* of each of the bodies within the system at any time. In general, both coordinates of *translation* and *rotation* are needed to describe the position of a rigid body. Taken all together, the coordinates are called *generalized coordinates*. For the *simple pendulum* shown, x_G and y_G are *coordinates of translation* that describe the *location* of the mass center of the bar, and θ is a *coordinate of rotation* that describes the *orientation* of the bar.

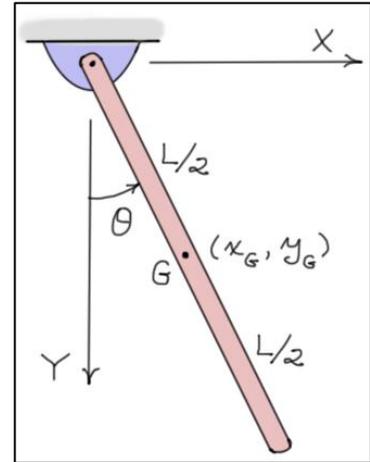


Fig. 1 Simple Pendulum

Typically, the *generalized coordinates* used to define the *configuration* of the mechanical system form a *dependent* set. That is, the coordinates *cannot all be chosen independently* of the others. For example, for the simple pendulum of Fig.1, x_G , y_G , and θ form a *dependent set* of coordinates. The following equations can be used to relate the three.

$$\boxed{x_G = \frac{L}{2} \sin(\theta)} \quad \boxed{y_G = \frac{L}{2} \cos(\theta)}$$

Given the *value* of *one* of the *coordinates*, these equations can be used to compute the *values* of the *other two coordinates*. So, only *one* of the coordinates is needed to describe the position of the system. *Any pair* of these coordinates forms a *dependent set*.

Generalized Coordinates and Degrees of Freedom

The number of *degrees of freedom* (DOF) of a mechanical system is defined as the *minimum number* of *generalized coordinates* necessary to define the configuration of the system. For a set of generalized coordinates to be *minimum* in *number*, the coordinates must form an *independent set*. Fig. 2 below shows *examples* of one, two, and three degree-of-freedom *planar systems*.

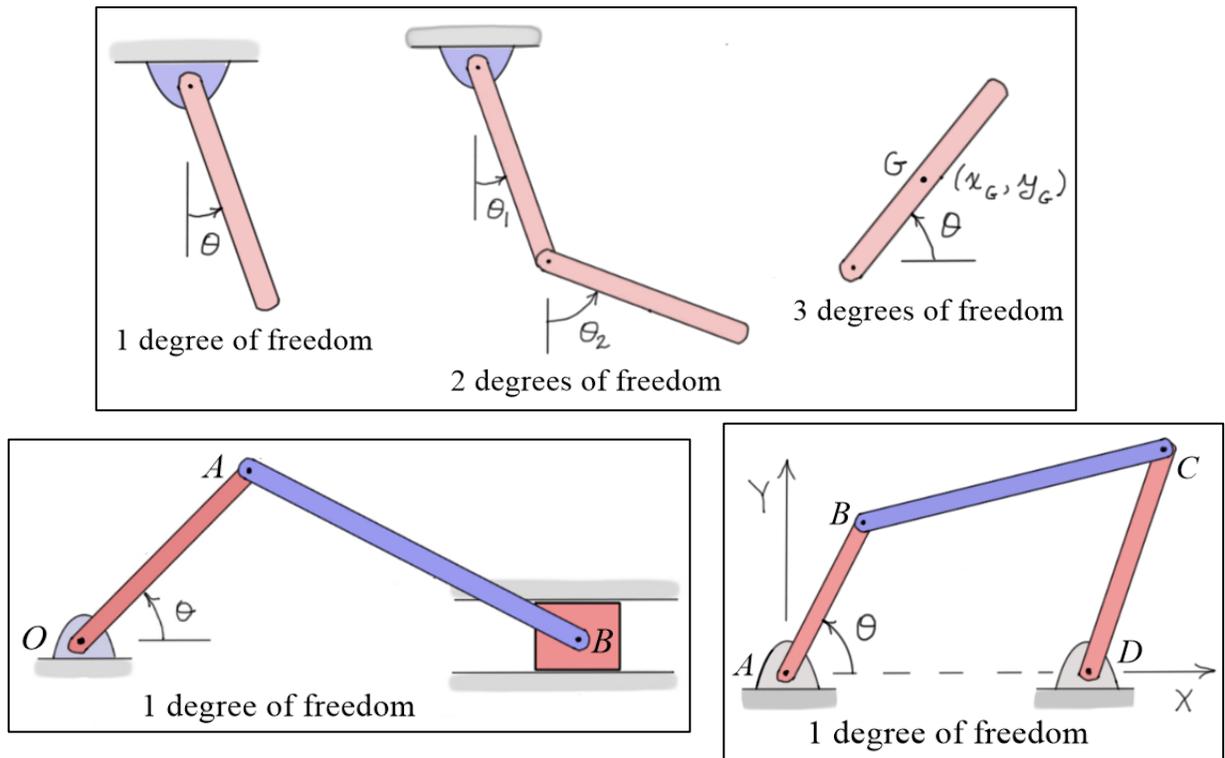


Fig. 2 Examples of One, Two, and Three Degree of Freedom Planar Systems

For *mechanical systems* that consist of a series of *interconnected bodies*, it may not be obvious *how many degrees of freedom* the system possesses. For these systems, the number of degrees of freedom can be found by first *calculating* the number of degrees of freedom the system would possess if the motions of all the bodies were *unrestricted* and then *subtracting* degrees of freedom *removed* by the *constraints* on the motion. For example, for the slider crank mechanism shown in Fig. 2, the number of degrees of freedom can be calculated in the following ways:

a) Counting 3 Bodies: (crank, slider, and connecting bar)

3 Bodies @ 3 DOF	$= 3 \times 3$	$= 9$ DOF (possible)
3 Pin Joints	$= 3 \times 2$	$= 6$ constraints
1 Slider Joint	$= 1 \times 2$	$= 2$ constraints
Total DOF	$= 9 - 6 - 2$	$= 1$ DOF

b) Counting 2 Bodies: (crank and connecting bar)

2 Bodies @ 3 DOF	$= 2 \times 3$	$= 6$ DOF (possible)
2 Pin Joints	$= 2 \times 2$	$= 4$ constraints
1 Pin/Slider Joint	$= 1 \times 1$	$= 1$ constraint
Total DOF	$= 6 - 4 - 1$	$= 1$ DOF