

Intermediate Dynamics

Natural Frequencies and Mode Shapes

To calculate the *natural frequencies* and *mode shapes* for *multiple degree-of-freedom* (DOF), rigid-body systems, the equations of motion (EOM) must first be *linearized* about some equilibrium (or steady-state) position and expressed in the following matrix form.

$$\boxed{[M]\{\Delta\ddot{q}\} + [C]\{\Delta\dot{q}\} + [K]\{\Delta q\} = \{F(t)\}}$$

Here, $[M]$, $[C]$, and $[K]$ represent the system's *mass*, *damping*, and *stiffness matrices*, vector $\{\Delta q\}$ represents *changes* in all the *generalized coordinates* from their equilibrium values, and vector $\{F(t)\}$ represents all the forces acting on the system. Setting the *damping matrix* and *force vector* to *zero* gives an equation of motion representing *free, undamped response*.

$$\boxed{[M]\{\Delta\ddot{q}\} + [K]\{\Delta q\} = \{0\}} \quad (\text{free, undamped response})$$

Following the pattern for single DOF systems, look for a solution of the following form.

$$\boxed{\{\Delta q\} = e^{j\omega t} \{u\}}$$

This equation describes a *steady-state, undamped* solution, with $\{u\}$ representing the *mode shape* of the oscillations. Substituting this into the differential EOM gives

$$\boxed{([K] - \omega^2[M]) \{u\} = \{0\}}$$

The problem now is to find ω^2 and $\{u\}$ that satisfy this *algebraic* equation. This is called an *eigenvalue problem*.

For this equation to have a *non-zero* solution for $\{u\}$, the *determinant* of the coefficient matrix must be *zero*. That is,

$$\boxed{\det([K] - \omega^2[M]) = 0}$$

If the matrices $[M]$ and $[K]$ are $N \times N$ matrices, the solution to this equation yields N values for ω^2 , and hence, N values for ω . These are the N *natural frequencies* of the system. Associated with each of these frequencies is a *mode shape* $\{u\}$. The N *eigenvalues* of the system are represented by ω^2 , and the N *eigenvectors* of the system are represented by the vector $\{u\}$.